

Dubai-I

“Voor en Door Staalconstructeurs”

23-11-2016

ir. R. van der Geer & ir. S. Maljaars

Westpoint Building 2 | De Brauwweg 60 | 3125 AE Schiedam | The Netherlands | www.kci.nl | info@kci.nl

KCI Company profile



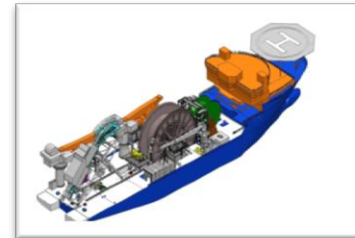
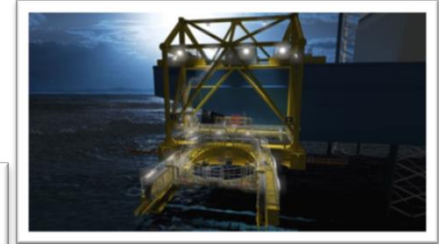
- Founded in 1987
- Main markets are
 - Oil & Gas
 - Renewables
 - Equipment
 - Wheels
- ISO 9001 certified
- FPAL registered no. 10049367
- Part of Oceanteam group

*“It’s our drive to create the best solutions
for our customers assets”*



Our Market Segments

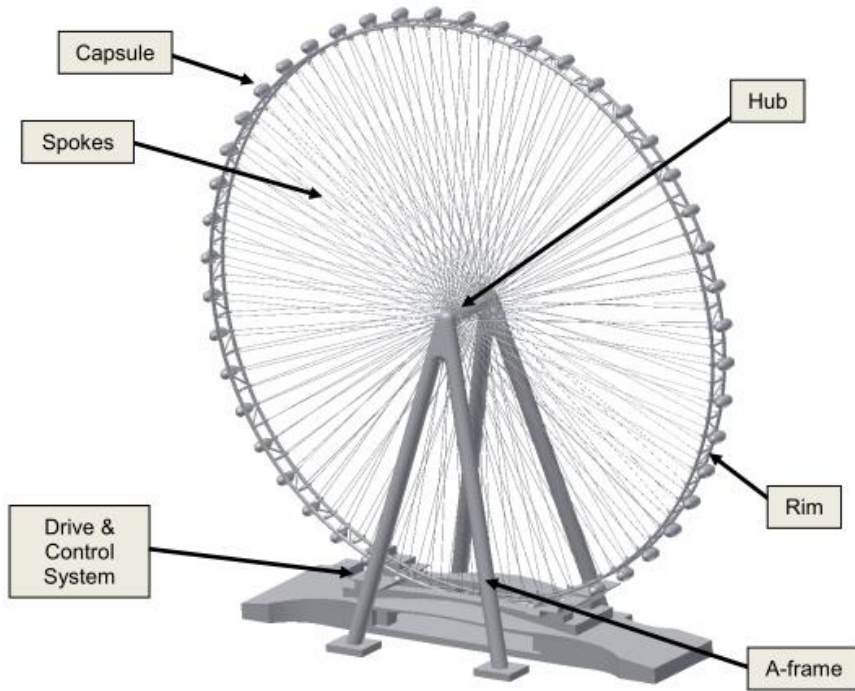
- **Renewables / Offshore Wind**
 - Substations (topside & foundations)
 - WTG foundations
 - Cable lay solutions
- **Oil & Gas**
 - Field development
 - Platforms, foundations & pipelines
- **Equipment**
 - Specialized equipment design for lifting, pipe & cable lay
 - Transport & Installation Engineering
 - Transport and lay analyses
 - Procedures
- **Wheels**
 - Design & engineering of observation wheels



1

General Project Info

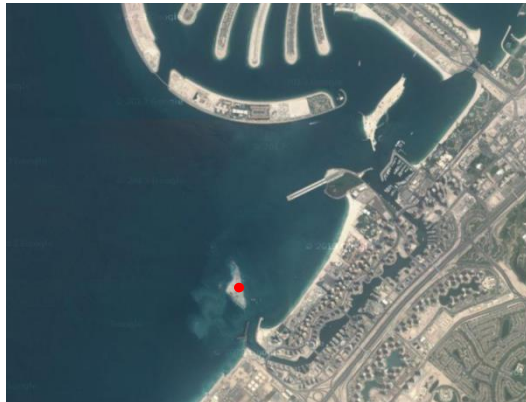
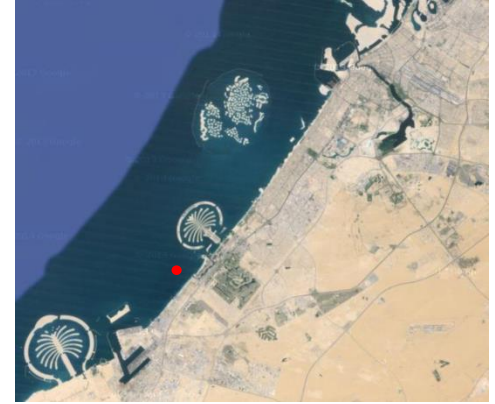
General Project Info – Introduction



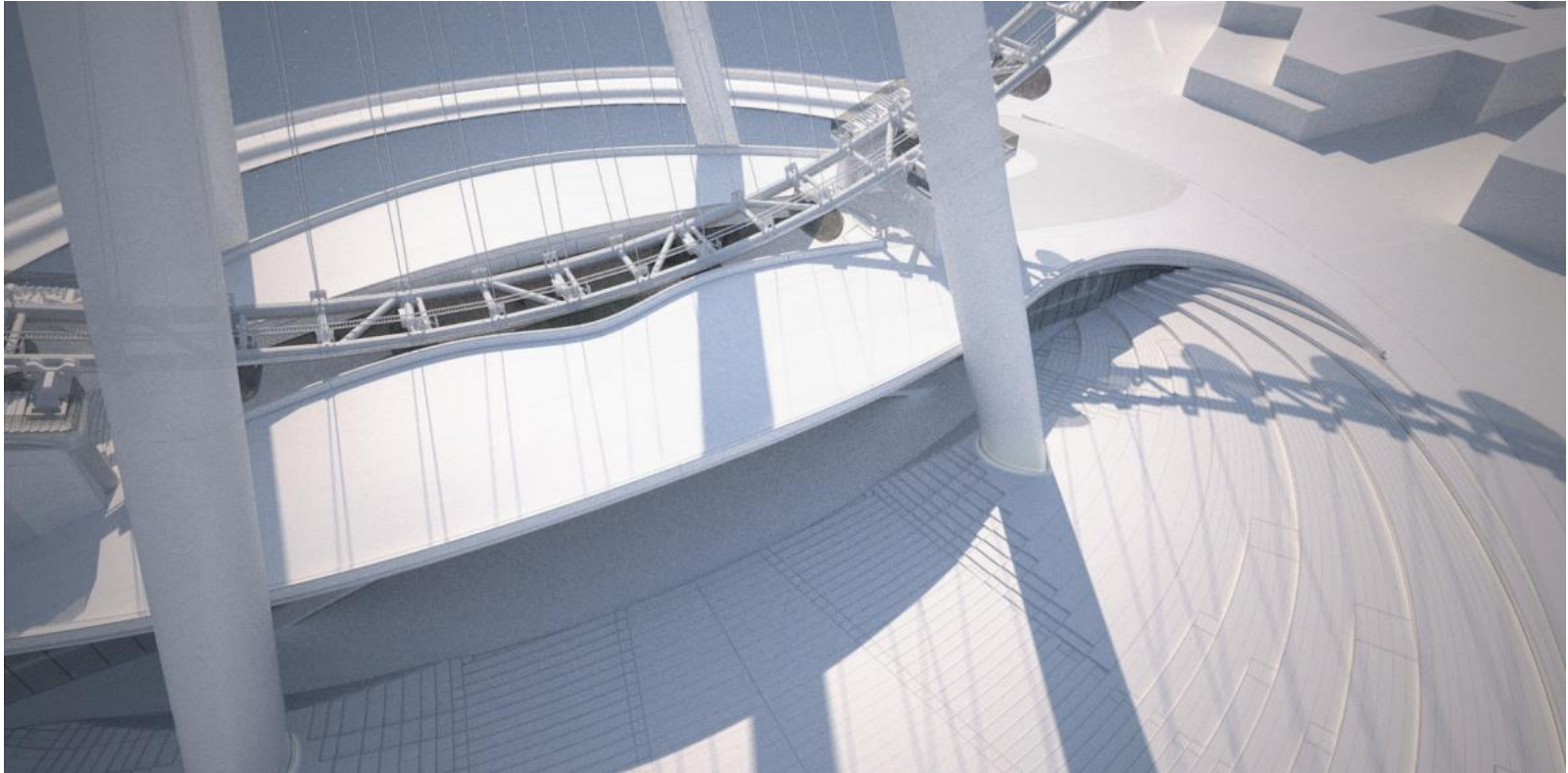
Key Figures:

- Approx. 250m Top Elevation
- 137.5m Spindle Elevation
- 238m Rim Diameter
- 48 Capsules (40p. Capacity Each)
- 192 Spoke Cables
- 4 Towers with Drive & Guide Units
- Total Mass approx. 11,000mton

General Project Info – Project Location



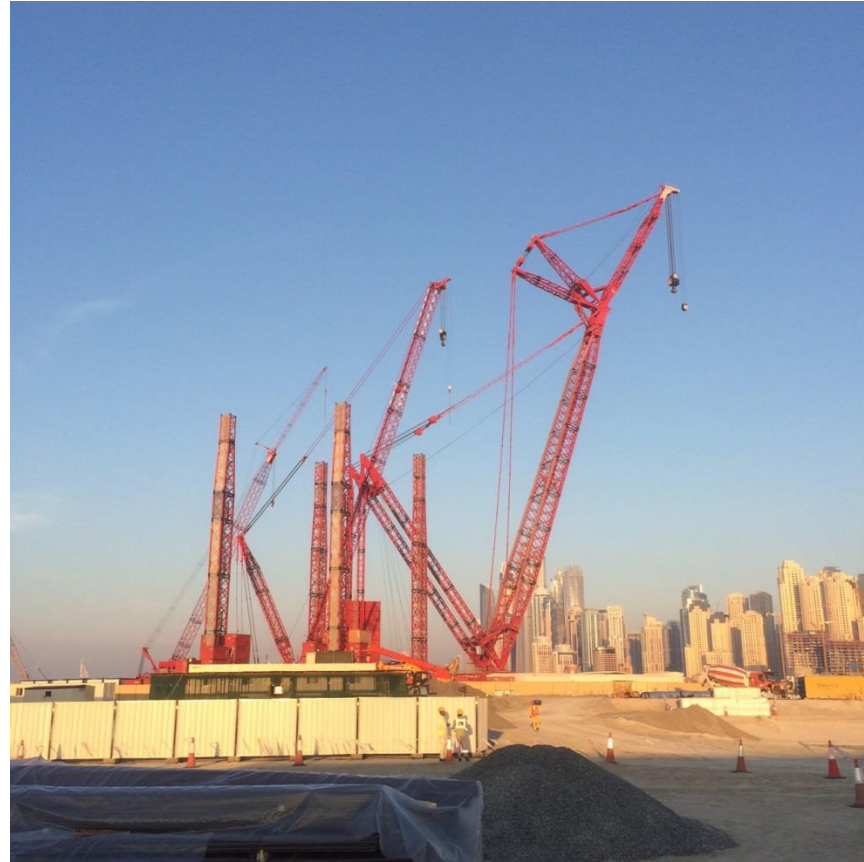
General Project Info – Artist impressions



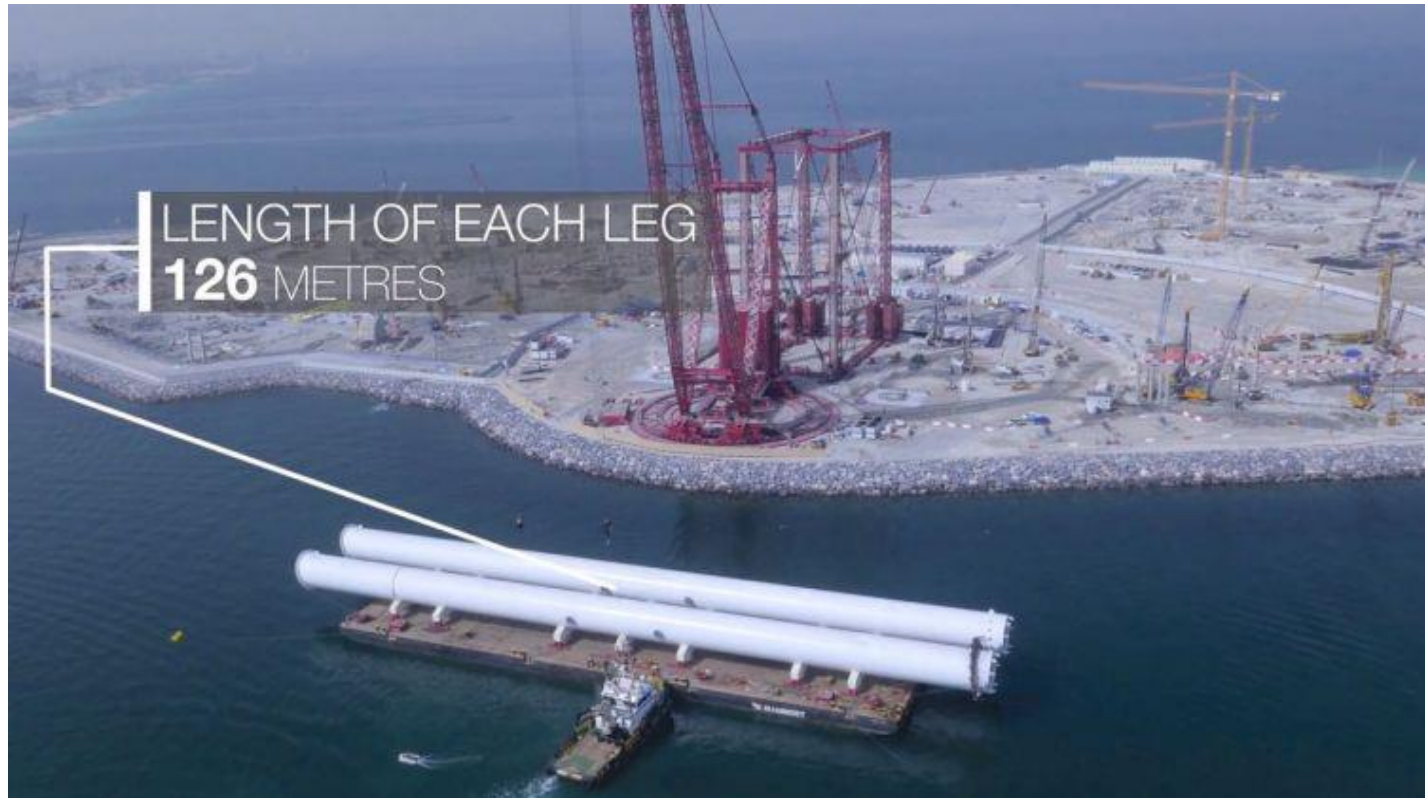
General Project Info – Artist impressions



General Project Info – Installation Status



General Project Info – Installation Status



General Project Info – Installation Status



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General Project Info – Installation Status



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Specific Engineering & Design Topics



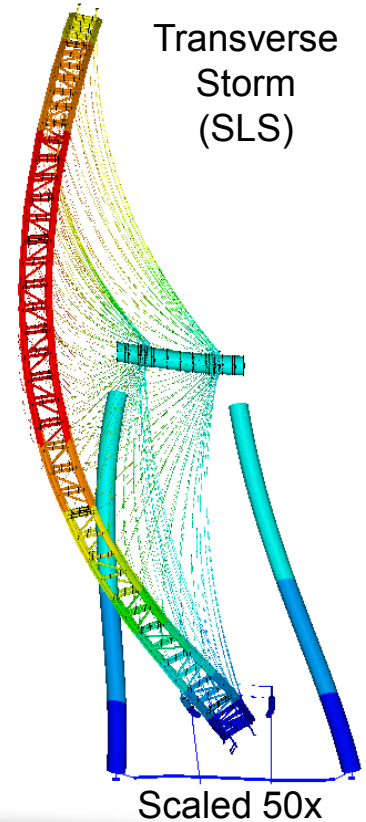
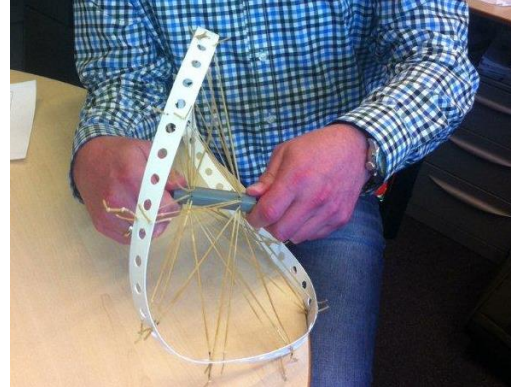
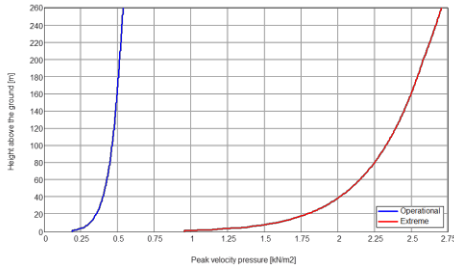
- **Static Analyses - Wind Load**
- **Dynamic Wind Response**
- ANSYS Post-Processing Routines
- Static Wind Tunnel Tests
- Dynamic Wind Tunnel Tests
- Spectral Seismic Analyses
- Seismic Time-History Analyses
- **Various Fatigue Analyses**
- Various Detailed Analyses
- SCF Calculations
- Capsule Design
- Evacuation Strategy
- Guide/Drive Unit Design
- Storm Lock Design
- Collector Gear Design
- Temperature Effects
- Rim Pushover Analyses
- A-Frame Pushover Analyses
- A-Frame Footing Design
- Imperfections
- Installation Analyses
- Bearing Design
- Spoke Cable Design
- Cable Socket Design
- Bent Limiter Design
- Cable Damper Design
- Vortex Induced Vibrations
- Tuned Mass Dampers
- Bearing Replacement
- Bearing Pad Exchange
- Bearing Tests
- Foundation Stiffness Variation
- Various Sensitivity Studies
- Restraint Tower Stiffness
- Hub-Drive Design
- Boarding Platform Gap Analyses
- Cable Replacement Analyses
- Accidental Spoke Cable Snap
- ... Many More

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Static Analyses

Static Analyses – Wind Load

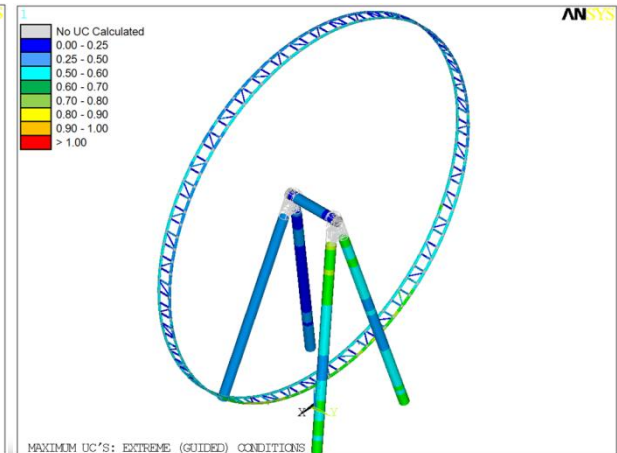
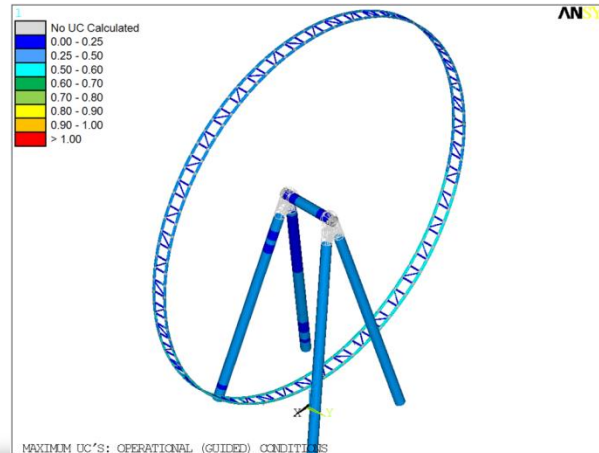
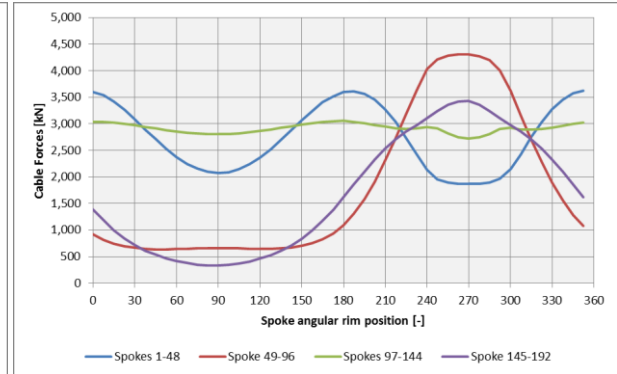
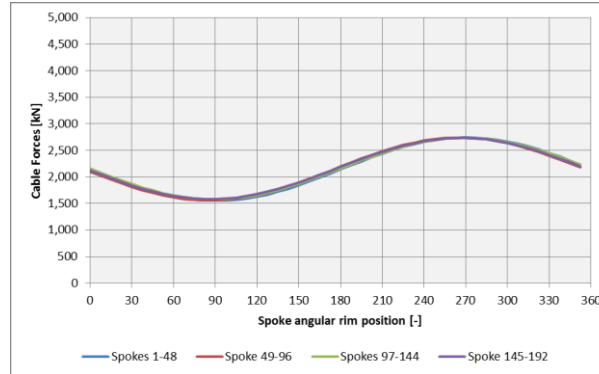
- Operational Wind Speed Limit : **20m/s** (3s gust)
- Extreme Wind Speed Limit : **45m/s** (3s gust)
- 5 Wind Directions
- 3 Wind Gust Distributions



Static Analyses – Monitored Parameters

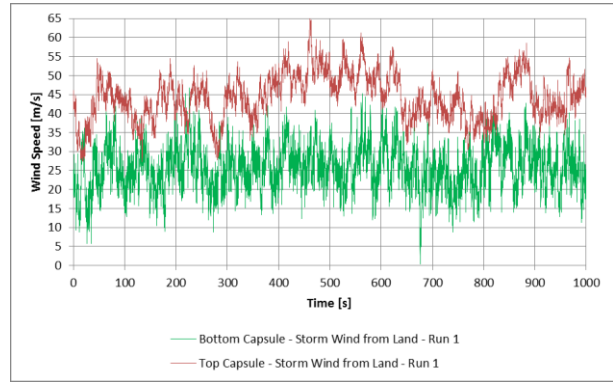
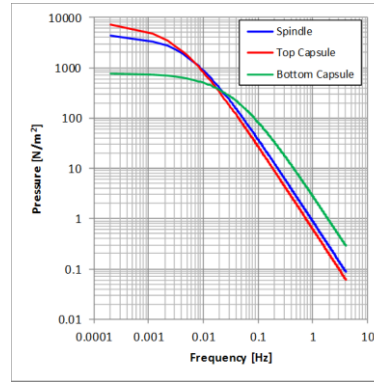
Monitored Results with User-Defined ANSYS routines:

- Cable Forces
- Member UC's
- Tubular Joint UC's
- Bearing Loads
- A-Frame Support Reactions
- Drive & Guide Forces
- Deflections
 - Spindle
 - A-Frame Foundations
 - Rim
 - Capsules at Boarding Platform

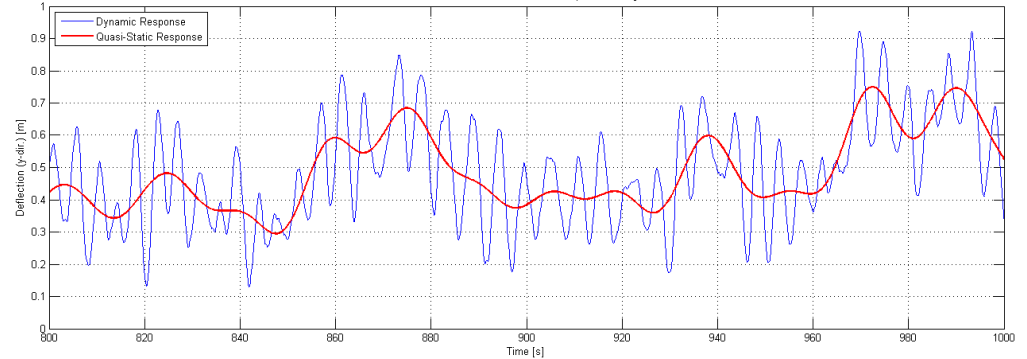


Dynamic Wind Response

- **Wind Response**
 - Time History Analysis
 - DAF-factor
 - Human Comfort



Storm Wind from Land - Run 1 - Capsule at Oddeg



3

Fatigue Analyses

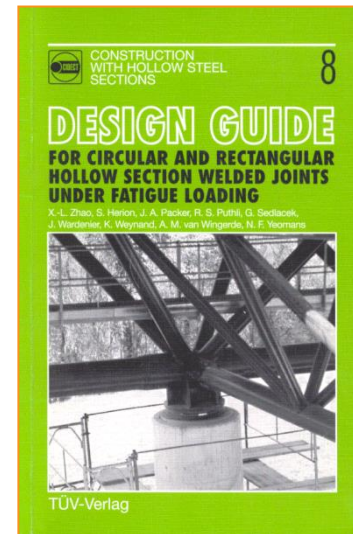
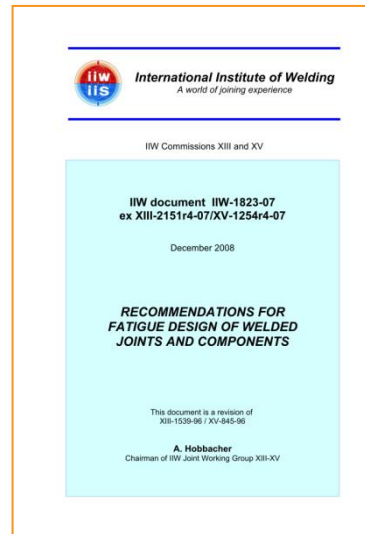
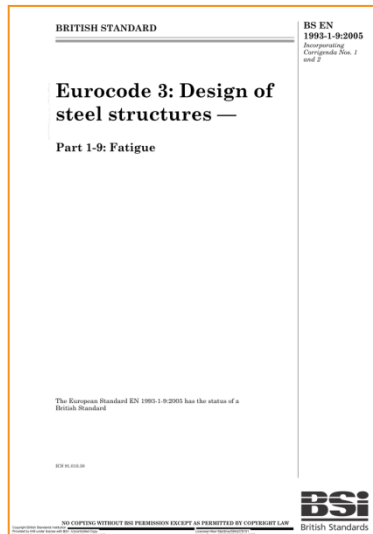
Fatigue Analyses - Introduction



- **Applied codes**
- **Fatigue loadings**
- **Applied methods**
- **Calculation examples**
- **Conclusions**

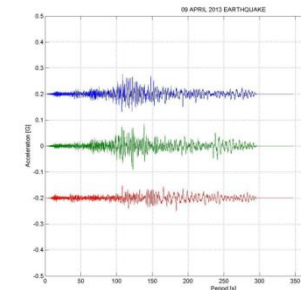
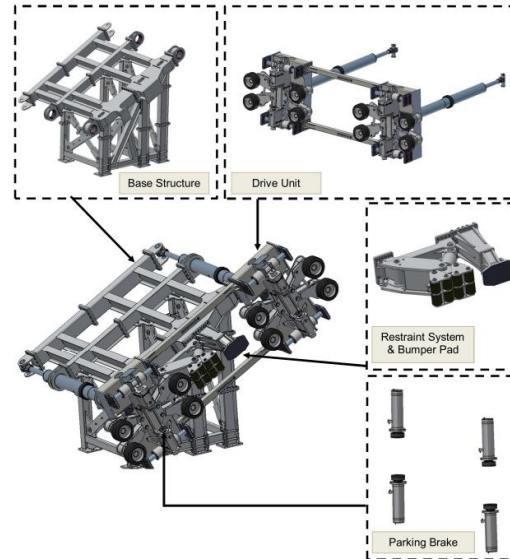
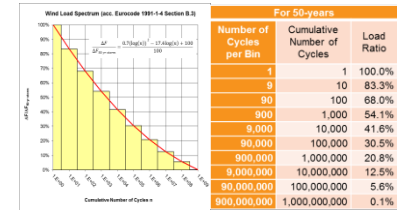
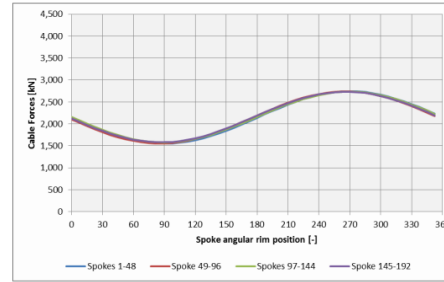
Fatigue Analyses – Applied codes

- Eurocode 3 - *BS EN 1993-1-9:2005*
- International Institute of Welding IIW - *document IIW-1823-07*
- DG-8-CIDECT - *Design guide for circular and rectangular hollow sections welded joints under fatigue loading*
- Several papers



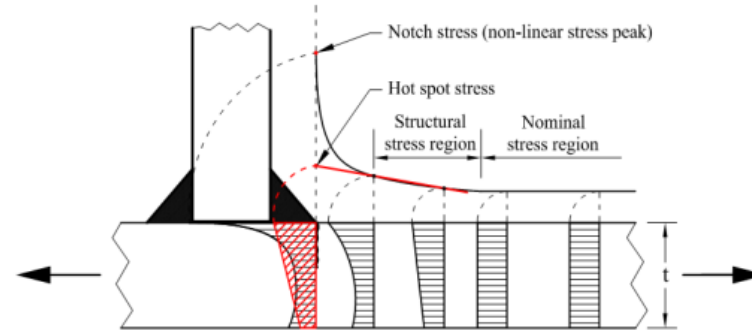
Fatigue Analyses – Fatigue loadings

- Wheel rotation (gravitational force)
- Interaction between rim drive box and drive and restraint system
- Wind effect
- Temperature effects
- Seismic activity



Fatigue Analyses – Applied methods

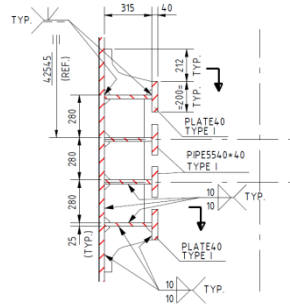
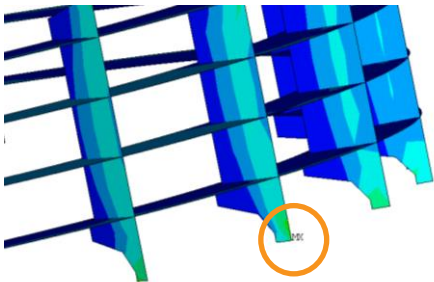
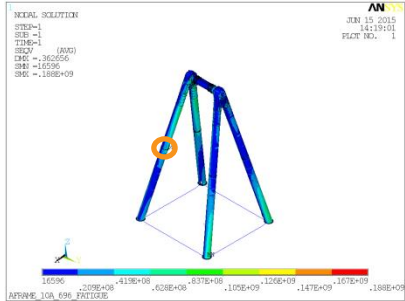
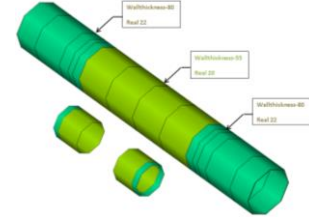
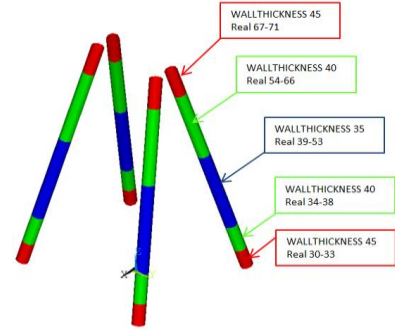
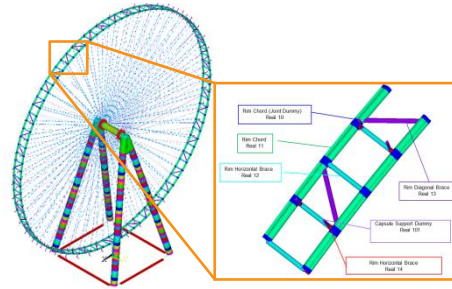
- Nominal stress method
- Hot spot stress method
- Notch stress method
- Fracture mechanics approach



Fatigue Analyses – Calculation examples

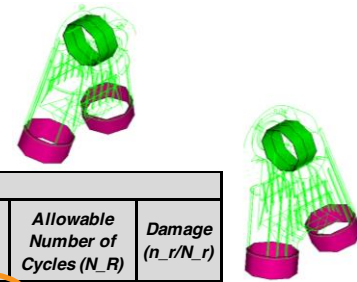


- **Nominal stress method**
 - Rim chord cross section
 - A-frame leg and spindle welds



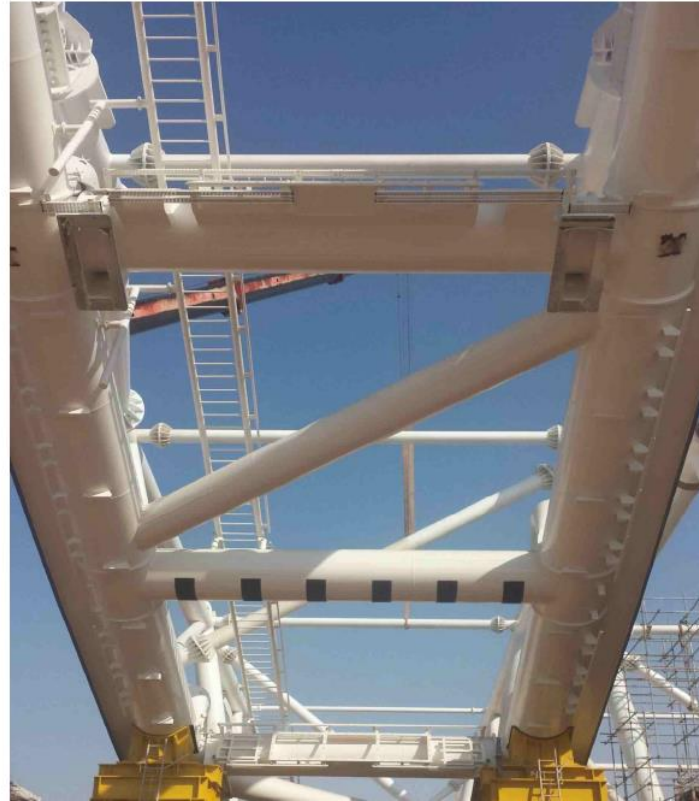
STRESS IN THE WELD DUE TO:	FX	T1.x	[MPa]	-65.28	
	FY	T2.y	[MPa]	0.82	
		Q2.y	[MPa]	N.A.	
	FZ	T2.z	[MPa]	N.A.	
		Q2.z	[MPa]	21.53	
	MX	T2.Mx	[MPa]	N.A.	
		Q2.Mx	[MPa]	0.62	
	MY	T2.My	[MPa]	N.A.	
		Q2.My	[MPa]	81.65	
	MZ	T2.Mz	[MPa]	1.70	
		Q2.Mz	[MPa]	N.A.	
	COMBINED STRESS		Tv1	[MPa]	2.52
		Ev2	[MPa]	-65.28	
		Qv1	[MPa]	103.80	
WF STRESSES		Twf	[MPa]	65.28	
		Qwf	[MPa]	103.83	

For 60-years					
Number of Cycles per Bin	Cumulative Number of Cycles	Load Ratio	Stress range ($\Delta\sigma$ R)	Allowable Number of Cycles (N_R)	Damage (n_r/N_r)
1	1	100.00%	104	1.76E+05	5.67E-06
9	10	85.00%	87	3.05E+05	2.95E-05
90	100	68.00%	71	5.61E+05	1.61E-04
900	1000	54.10%	56	1.11E+06	8.09E-04
9000	10000	41.60%	43	2.45E+06	3.68E-03
90000	100000	30.50%	32	7.18E+06	1.25E-02
900000	1000000	20.80%	22	4.87E+07	1.85E-02
9000000	10000000	12.50%	13	INFINITE	0.00E+00
90000000	100000000	5.60%	6	INFINITE	0.00E+00
9E+08	1000000000	0.10%	0	INFINITE	0.00E+00
				D_WIND	0.0357



Fatigue Analyses – Calculation examples

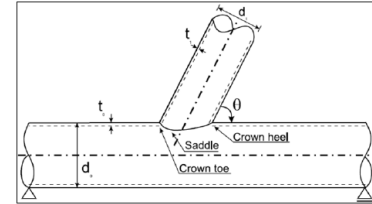
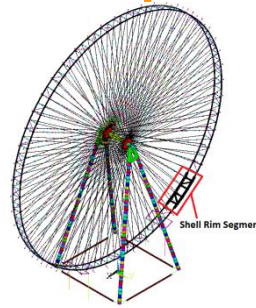
- **Nominal stress method**
 - Rim chord cross section
 - A-frame leg and spindle welds
 - Secondary steel attachments



Fatigue Analyses – Calculation examples

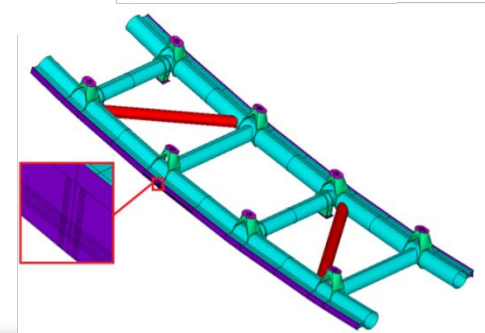
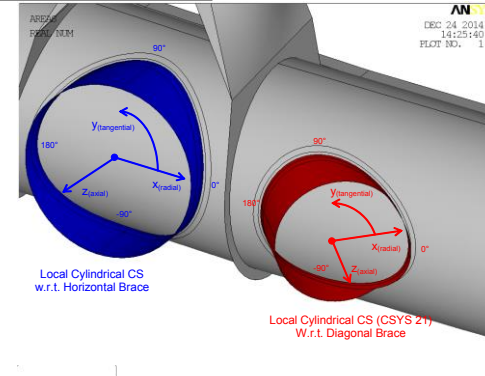
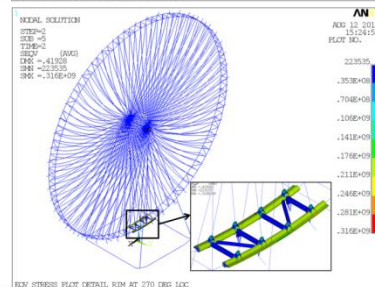
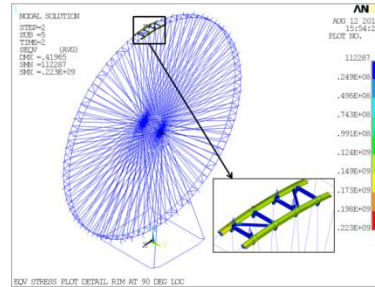
- **Nominal stress method**

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- Secondary steel attachments



- **Hot spot stress method**

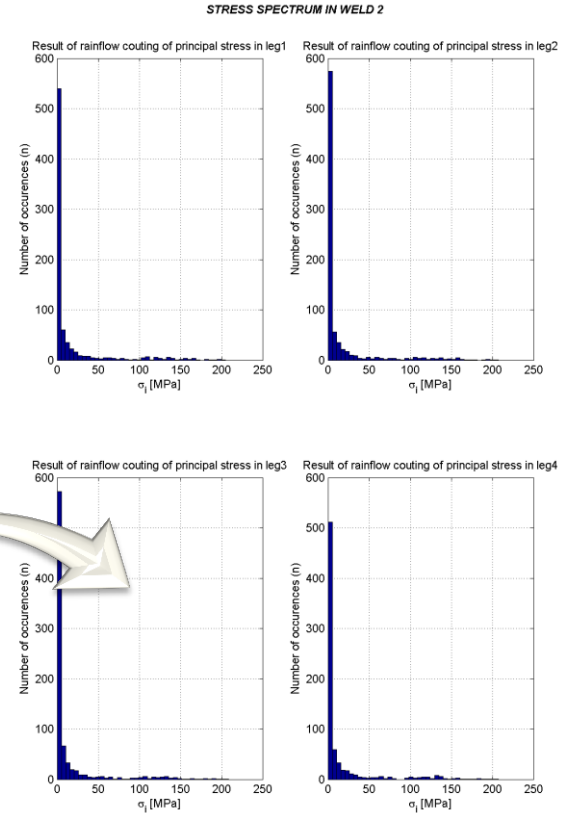
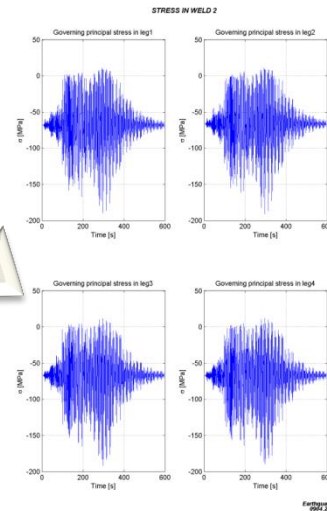
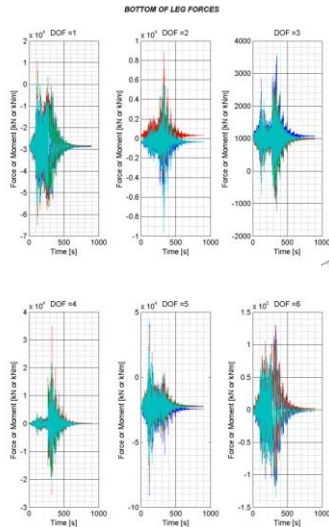
- Rim chord tubular joints (SCF functions CIDECT)
- Rim chord tubular joints (hot spot stress according to CIDECT)
- Local details in rim (hot spot stress according IIW)
- Weld details in A-frame and Brace



Fatigue Analyses – Calculation examples

Weld details in A-frame and Brace

- Fatigue due to a seismic event (time domain analysis)
- Member loads are calculated (beam model)
- Hot spot stress signal over time
- Rainflow counting applied on the stress signal
- Miner rule for damage accumulation



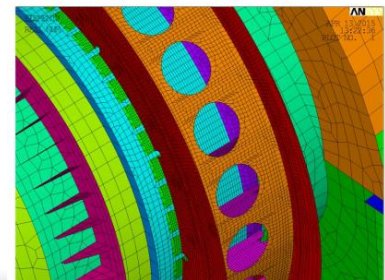
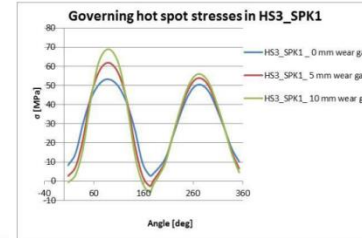
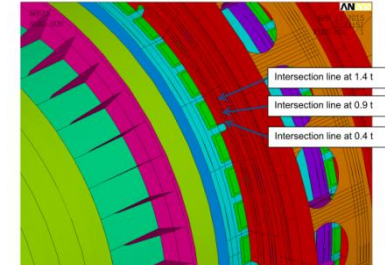
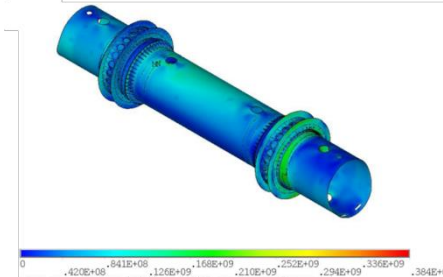
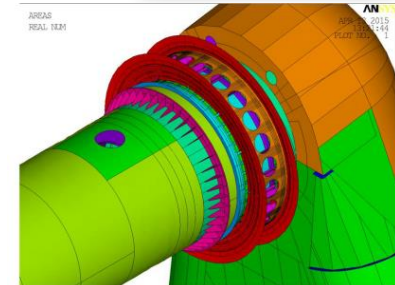
Fatigue Analyses – Calculation examples

- **Nominal stress method**

- Rim chord cross section
- A-frame leg and spindle welds
- Secondary steel attachments

- **Hot spot stress method**

- Rim chord tubular joints (SCF functions CIDECT)
- Rim chord tubular joints (hot spot stress according to CIDECT)
- Local details in rim (hot spot stress according IIW)
- Weld details in A-frame and Brace
- All welds in the rotating hubs



Fatigue Analyses – Calculation examples

- **Nominal stress method**
 - Rim chord cross section
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 - Secondary steel attachments
- **Hot spot stress method**
 - Rim chord tubular joints (SCF functions CIDECT)
 - Rim chord tubular joints (hot spot stress according to CIDECT)
 - Local details in rim (hot spot stress according IIW)
 - Weld details in A-frame and Brace
 - All welds in the rotating hubsnubs
- **Notch stress method**
 - Conical spoke to rim joint



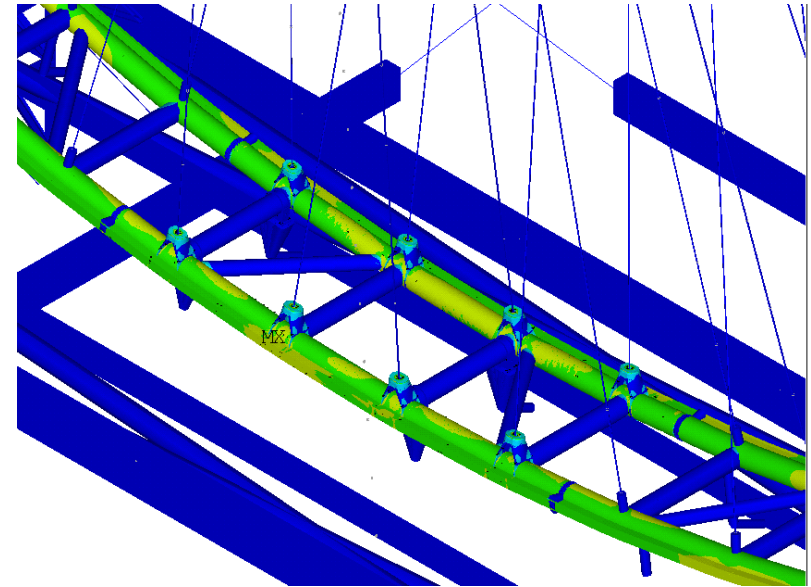
Fatigue Analyses – Calculation examples

Conical spoke to rim joint

1. Beam/shell model



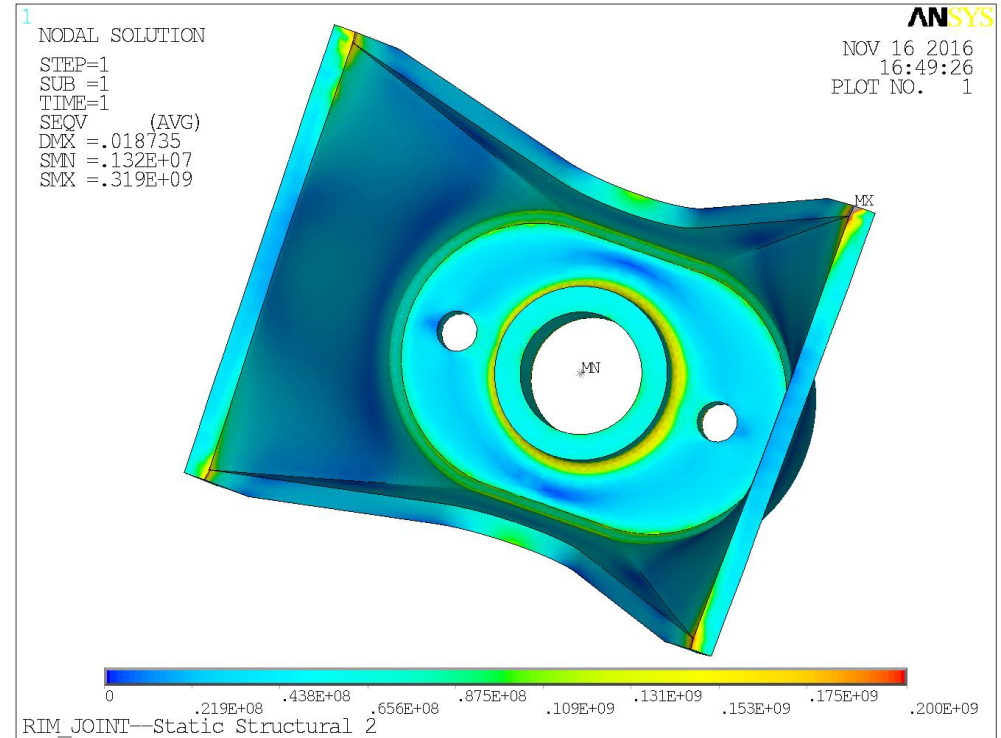
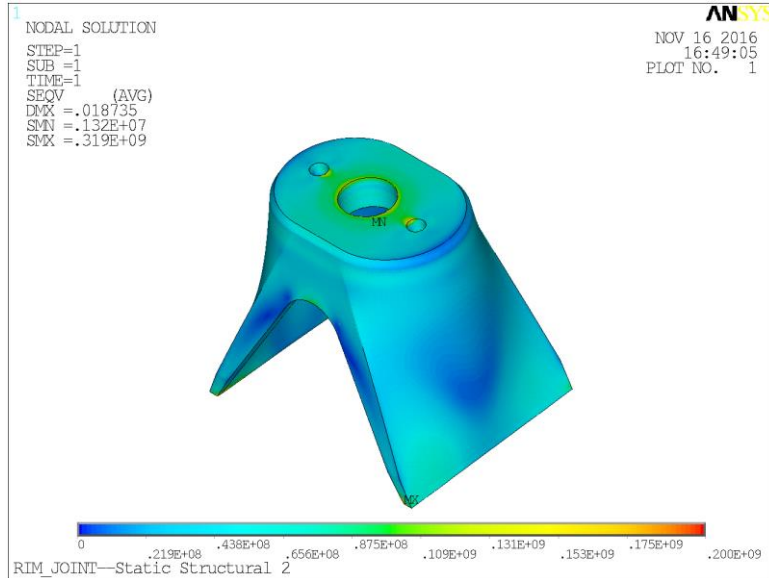
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ANSYS 11.0SP1  
NOV 16 2016  
16:46:15  
PLOT NO. 1  
NODAL SOLUTION  
STEP=1  
SUB =1  
TIME=1  
SEQV (AVG)  
PowerGraphics  
EFACET=1  
AVRES=Mat  
DMX =.405719  
SMX =.332E+09  
0  
.369E+08  
.738E+08  
.111E+09  
.148E+09  
.185E+09  
.222E+09  
.258E+09  
.295E+09  
.332E+09
```



Fatigue Analyses – Calculation examples

Conical spoke to rim joint

2. Solid sub-model

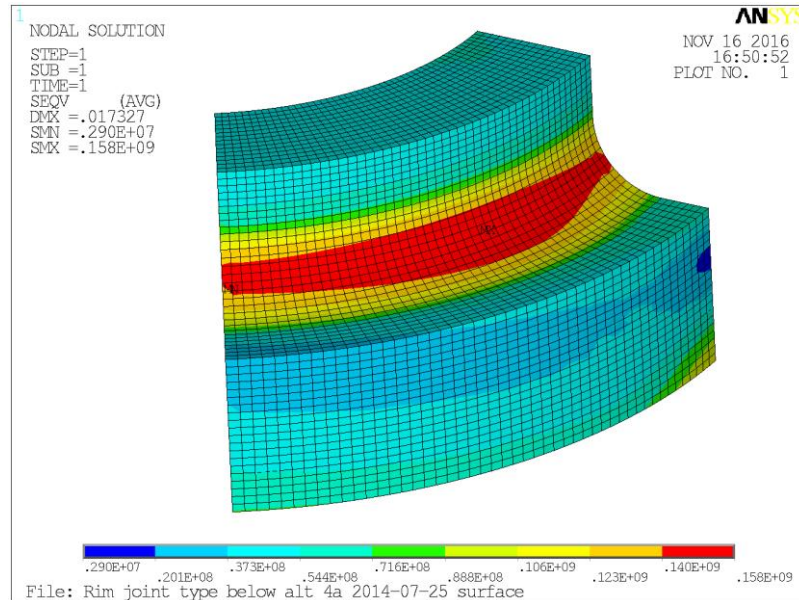


Fatigue Analyses – Calculation examples

Conical spoke to rim joint

3. Solid sub-sub-model

- Grinded weld is modeled ($R=35\text{mm}$), FAT 112 is used
- Principal stress is used



Fatigue Analyses – Calculation examples

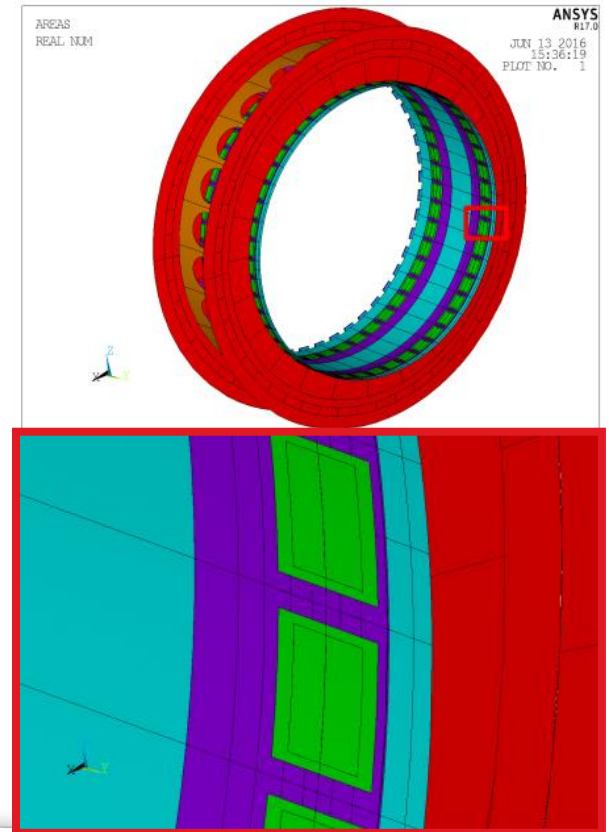
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 - Local details in rim (hot spot stress according IIW)
 - Weld details in A-frame and Brace
 - All welds in the rotating hubsnubs
- **Notch stress method**
 - Conical spoke to rim joint
 - Weld details in rotating hubs



Fatigue Analyses – Calculation examples

Weld details in rotating hubs

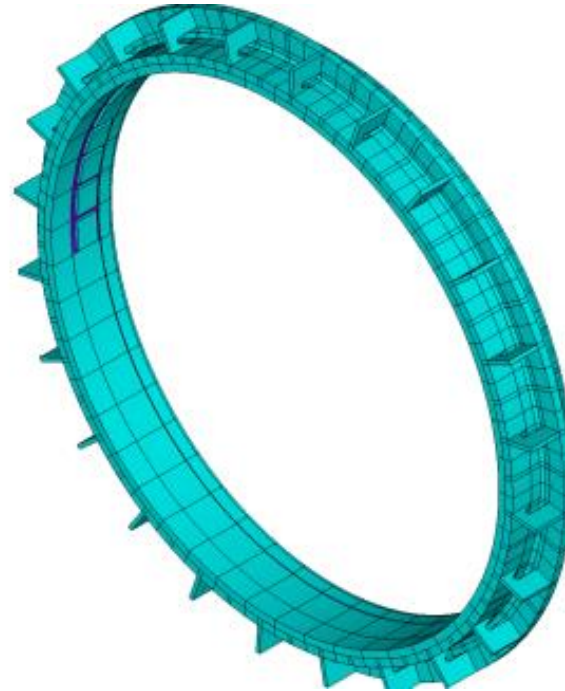
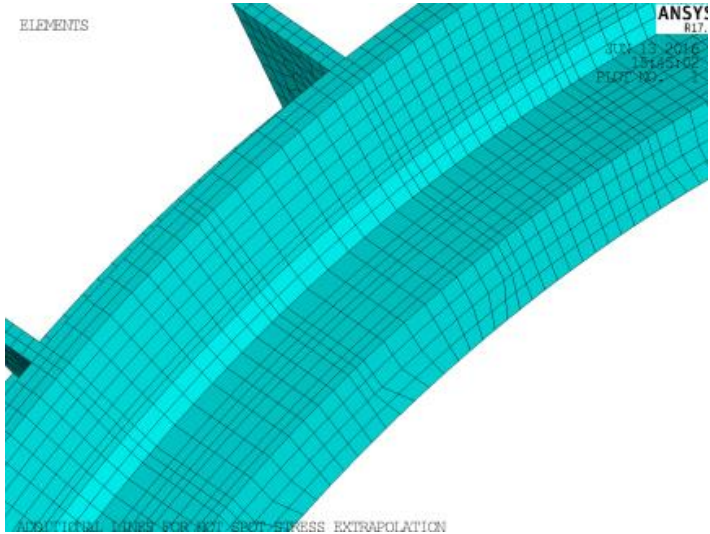
1. Shell model



Fatigue Analyses – Calculation examples

Weld details in rotating hubs

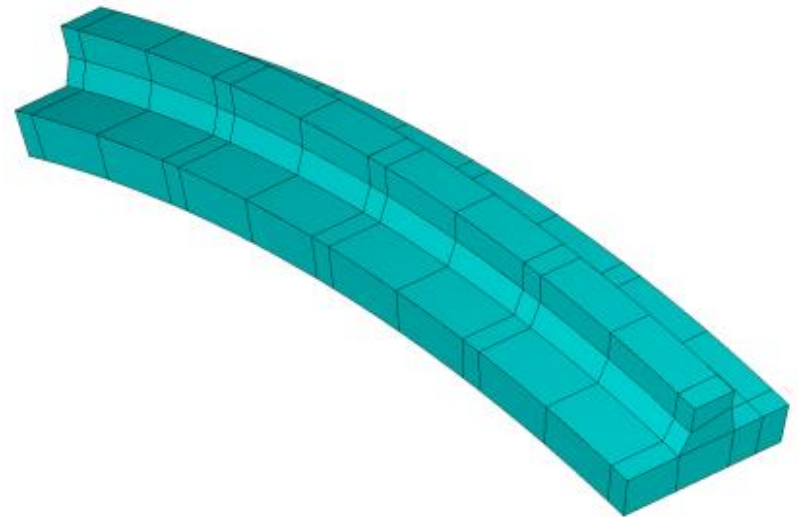
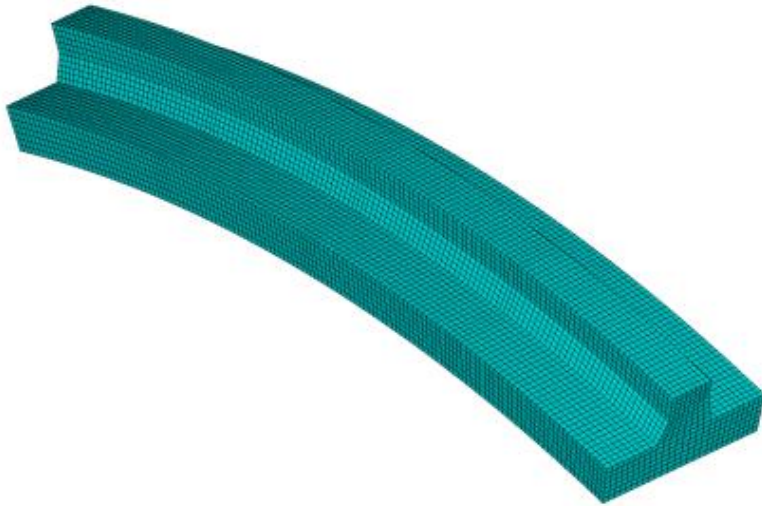
2. Solid sub-model



Fatigue Analyses – Calculation examples

Weld details in rotating hubs

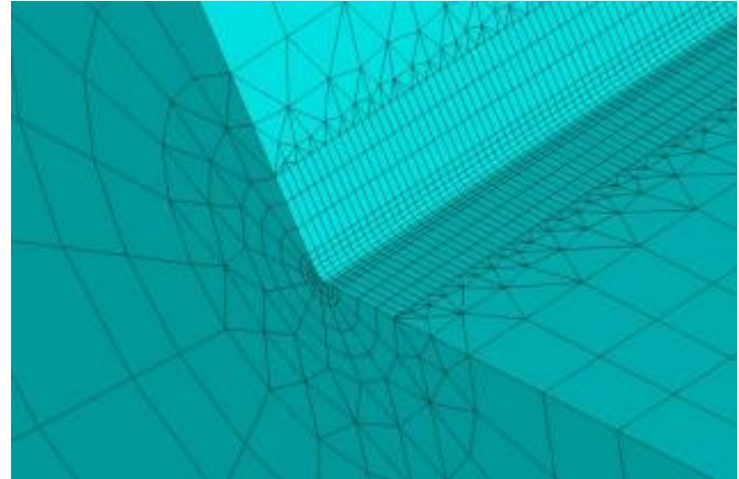
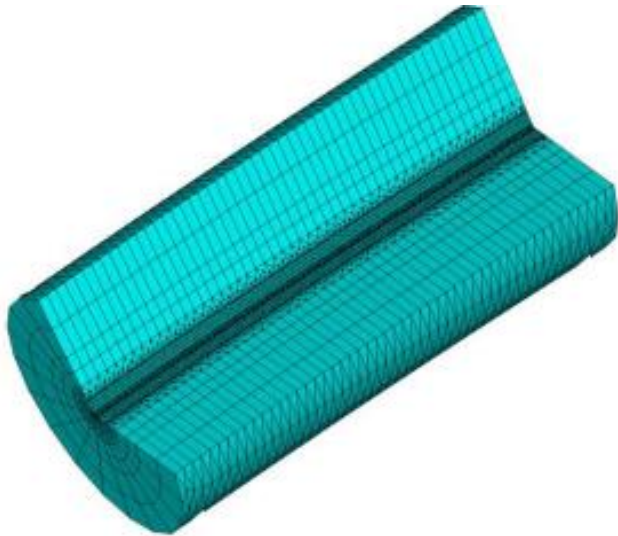
3. Solid sub-sub-model



Fatigue Analyses – Calculation examples

Weld details in rotating hubs

4. Solid sub-sub-sub-model



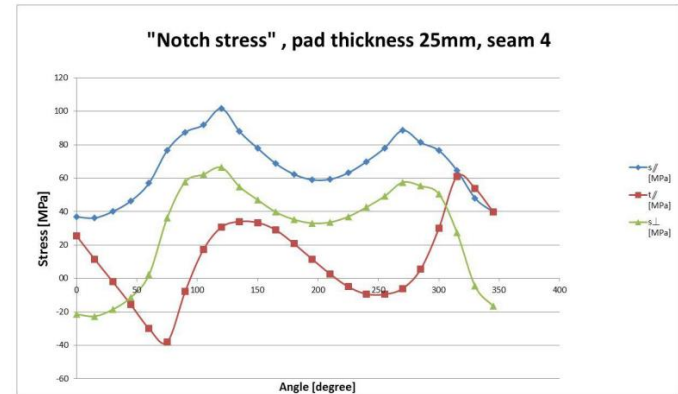
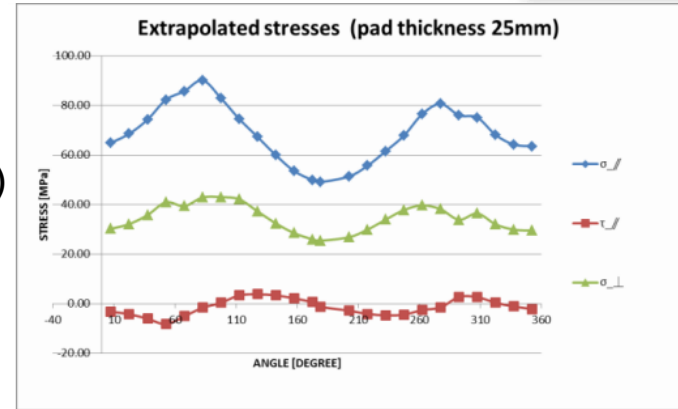
Fatigue Analyses – Calculation examples

Weld details in rotating hubs

5. Fatigue calculation

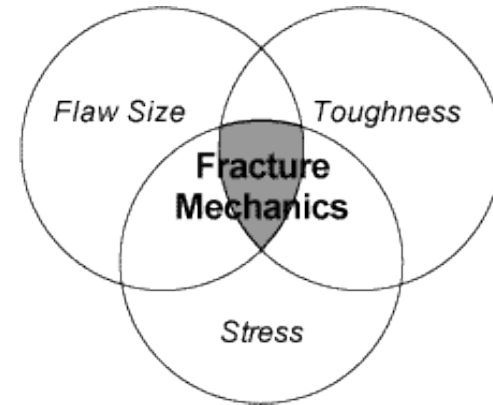
- Result of 2 typical weld details (as-welded)
- Result of 2 typical weld details (grinded)
- Which outcome is correct?

Weld detail	HSS / NSA (as-welded)	HSS / NSA (grinded)
1)	250	55
2)	32	10



Fatigue Analyses – Calculation examples

- **Nominal stress method**
 - Rim chord cross section
 - A-frame leg and spindle welds
 - Secondary steel attachments
- **Hot spot stress method**
 - Rim chord tubular joints (SCF functions CIDECT)
 - Rim chord tubular joints (hot spot stress according to CIDECT)
 - Local details in rim (hot spot stress according IIW)
 - Weld details in A-frame and Brace
 - All welds in the rotating hub
- **Notch stress method**
 - Conical spoke to rim joint
 - Weld details in rotating hubs
- **Fracture mechanics approach**
 - Currently under consideration



$$K = Y\sigma\sqrt{\pi a}$$

Fatigue Analyses – Conclusions

- **Nominal stress approach**
 - Easy to use in beam/tubular structures
 - Relative quick lifetime estimation for standard weld details
- **Hot spot stress approach**
 - Predict fatigue in geometric complex structures
 - Fatigue predictions in (relative) thin walled structures
 - Be careful while using this method in very thick walled structures
- **Notch stress approach**
 - Can be used at machined weld details
 - Can be used for non-categorized weld details



Thank you for your attention!

